Analysis of Axisymmetric Crushing of Frusta

M.M.A. Khan and H. Abbas

Aligarh Muslim University, Aligarh – 202 002

and

N.K. Gupta

Indian Institute of Technology Delhi, New Delhi - 110 016

ABSTRACT

The paper presents a curved-fold model with variable straight length for the axisymmetric crushing of thin frusta. The folding considered in the model is partly inside and partly outside. The variation of circumferential strain during the formation of a fold has been taken into account. The size of the fold and mean, as well as variation of crushing load, has been computed mathematically. The study is purely analytical and does not involve any empirical constant; and hence, can be used in general. The model's predictions have been compared with experimental results and a reasonably good agreement has been observed.

Keywords: Thin-walled structures, energy absorbers, cylindrical tubes, frusta, energy absorbing device, curved-fold model, axisymmetric crushing

NOMENCLATURE

 θ Angle of taper of the frusta

 f_{v} Yield stress of material

- t Thickness of tube
- α , β Angles subtended by arc AB (or B'C) and CC' (or D'E) at their respective centres
- ρ, σ Radii of curvature in lower and upper limbs in a fold, respectively
- α_1, β_1 Angles subtended by arcs AA' and D'E at their respective centres
- D_1, D_2 End diameter of tube
- *m* Folding parameter

- L, L' Fold length of lower and upper limbs, respectively
- R, R' Mean radii of frustum at A' and D', respectively
- aL, aL' Length of straight portion in lower and upper limbs, respectively
- *bL*, *bL*' Length of curved portion in lower and upper limbs, respectively
- *cL*, *cL*' Length of one straight and one curved portion in lower and upper limbs, respectively
- ρ_m, ρ_m' Final radii of curvature in lower and upper limbs in a fold, respectively
- α_m Maximum hinge Angle

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Figure 1. Model for crushing of frusta for case1 (case 1: m lies between 0 and b).

Incremental Energy Absorbed in Bending of Lower Limb

Radii of tube at A and C are

$$R_{A} = R + \rho \left\{ \cos \left(\alpha_{1} + \theta \right) - \cos \theta \right\}$$
(10)

$$R_{C} = R + \rho \left\{ \frac{\cos\theta - 2\cos(\alpha + \theta)}{+\cos(\alpha_{1} + \theta)} \right\} + aL\sin(\alpha + \theta)$$
(11)

Energy to bend the hinges for a small increment in angle by $d\alpha$ is:

$$dW_b = 2\pi M_p (R_A + R_C) d\alpha \qquad (12)$$

Substituting the values of R_{λ} and R_{c} from Eqns (10) and (11), one obtains:

$$dW_{b} = 4\pi M_{p} \begin{bmatrix} R + \frac{bL}{\alpha} \begin{bmatrix} \cos\left(\alpha_{1} + \theta\right) \\ -\cos\left(\alpha + \theta\right) \end{bmatrix} \\ + \frac{aL}{2} \sin\left(\alpha + \theta\right) \end{bmatrix} d\alpha$$
(13)

Incremental Energy Absorbed in Hoop Action of Lower Limb

The expressions for incremental energy due to hoop strain for various regions have been derived as

Region AA'

$$dW_{1} = 2\pi t f_{y} L^{2} \left[\frac{\frac{m^{2} \sin(\alpha_{1} + \theta)}{\alpha} + \frac{mb \cos(\alpha_{1} + \theta)}{\alpha^{2}} + \frac{b}{\alpha^{3}} \left\{ \sin \theta - \sin(\alpha_{1} + \theta) \right\} \right] d\alpha$$
(14)

Region A'B

$$dW_{2} = 2\pi t f_{y} L^{2} \begin{bmatrix} \frac{(m-b)\sin(\alpha_{1}+\theta)}{\alpha} + \\ \frac{b}{\alpha^{2}} \left\{ m\cos(\alpha_{1}+\theta) \\ -b\cos(\alpha+\theta) \right\} \\ + \frac{b^{2}}{\alpha^{3}} \left\{ \sin(\alpha+\theta) \\ -\sin(\alpha_{1}+\theta) \right\} \end{bmatrix} d\alpha$$
(15)

Region BB'

$$dW_{3} = 2\pi t f_{y} L^{2} \begin{bmatrix} \frac{a}{\alpha} \left\{ b \sin(\alpha + \theta) \\ -m \sin(\alpha_{1} + \theta) \right\} \\ + \frac{a^{2}}{2} \cos(\alpha + \theta) \end{bmatrix} d\alpha$$
(16)

Region B'C

$$dW_{4} = 2\pi t f_{y} L^{2} \begin{bmatrix} \frac{b}{\alpha} \left\{ b \sin(\alpha + \theta) \\ -m \sin(\alpha_{1} + \theta) \right\} \\ +b^{2} \cos(\alpha + \theta) \\ -\frac{b^{2}}{\alpha^{2}} \cos(\alpha + \theta) \\ +\frac{b^{2}}{\alpha^{3}} \left\{ \sin(\alpha + \theta) - \sin\theta \right\} \end{bmatrix} d\alpha$$
(17)

• Total Incremental Energy Absorbed in Lower Limb

Total incremental energy for lower limb is obtained by adding Eqns (14) to (17), thus

$$dW_{T1} = \frac{\pi f_{y}t^{2}}{\sqrt{3}} \begin{bmatrix} 2R + \frac{2bL}{\alpha} \left\{ \cos\left(\alpha_{1} + \theta\right) \\ -\cos\left(\alpha + \theta\right) \right\} \\ + aL\sin\left(\alpha + \theta\right) \end{bmatrix} d\alpha$$
$$+ 2\pi f_{y}tL^{2} \left[\frac{(2m - 1)m}{\alpha} \sin\left(\alpha_{1} + \theta\right) \\ + \frac{bc}{\alpha} \sin\left(\alpha + \theta\right) + \frac{2}{\alpha^{2}} \left\{ \frac{mb\cos\left(\alpha_{1} + \theta\right)}{-b^{2}\cos\left(\alpha + \theta\right)} \right\} \\ + \frac{2b^{2}}{\alpha^{3}} \left\{ \sin\left(\alpha + \theta\right) - \sin\left(\alpha_{1} + \theta\right) \right\} \\ + (0.5a + b)\cos\left(\alpha + \theta\right) \end{bmatrix} d\alpha$$
(18)

Total energy absorbed in full crushing of lower

limb,
$$W_{T1} = \int_{0}^{\alpha} dW_{T1}$$
 or

$$W_{T1} = \frac{\pi f_y t^2}{\sqrt{3}} \begin{bmatrix} 2R\alpha_m + aL \begin{cases} \cos\theta \\ -\cos(\alpha_m + \theta) \end{cases} \\ + 2bLX^* \\ + 2\pi t f_y L^2 Y^* \end{cases}$$
(19)

where

$$X^* = \int_0^{\alpha_n} \frac{1}{\alpha} \left\{ \cos \left(\alpha_1 + \theta \right) - \cos \left(\alpha + \theta \right) \right\} d\alpha$$
(20)

$$Y^* = \int_{0}^{\alpha_m} \frac{1}{\alpha} \left\{ m(m-1)\sin(\alpha_1 + \theta) - b\sin(\alpha + \theta) \right\} a$$
$$+ \frac{b^2}{\alpha_m^2} \left\{ \sin(\alpha_{1m} + \theta) - \sin(\alpha_m + \theta) \right\}$$
$$+ \frac{b}{\alpha_m} \left\{ b\cos(\alpha_m + \theta) - m\cos(\alpha_{1m} + \theta) \right\}$$
$$\left(0.5a^2 + b^2 \right) \left\{ \sin(\alpha_m + \theta) - \sin\theta \right\}$$
(21)

where

$$\alpha_{1m} = \frac{m\alpha_m}{b} \tag{22}$$

• Total Energy Absorbed in Upper Limb

Total potential energy for upper limb can be obtained by multiplying α , α_1 , α_m , α_{1m} by e^2 and replacing L by hL and θ by $-\theta$ in the energy equation of lower limb. The final expression will thus be:

$$W_{T2} = \frac{\pi f_y t^2}{\sqrt{3}} \begin{bmatrix} 2Rh\alpha_m + ahL \begin{cases} \cos\theta \\ -\cos\left(e^2\alpha_m\right) \\ + 2bhLX_1^* \end{bmatrix} \\ + 2\pi tf_y h^2 L^2 Y_1^* \tag{23}$$

where

$$X_{1}^{*} = \int_{0}^{\alpha_{n}} \frac{1}{\alpha} \left\{ \cos\left(e^{2}\alpha_{1} - \theta\right) - \cos\left(e^{2}\alpha - \theta\right) \right\} d\alpha$$
(24)

$$Y_{1}^{*} = \int_{0}^{\alpha_{m}} \frac{1}{\alpha} \begin{cases} m(m-1)\sin(e^{2}\alpha_{1}-\theta) \\ -b\sin(e^{2}\alpha-\theta) \end{cases} d\alpha + \frac{b^{2}}{e^{4}\alpha_{m}^{2}} \\ \begin{cases} \sin(e^{2}\alpha_{1m}+\theta) - \sin(\alpha_{m}+e^{2}\theta) \\ e^{2}\alpha_{m} \end{cases} \\ \begin{cases} b\cos(e^{2}\alpha_{m}-\theta) - m\cos(e^{2}\alpha_{1m}-\theta) \\ + (0.5a^{2}+b^{2}) \\ sin(e^{2}\alpha_{m}+\theta) + sin\theta \end{cases} \end{cases}$$
(25)

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• Total Energy Absorbed in Formation of One Complete Fold

The total energy absorbed in the formation of one complete fold will thus be:

$$W_{T} = W_{T1} + W_{T2}$$

$$= \frac{\pi f_{y} t^{2}}{\sqrt{3}} \begin{bmatrix} 2R\alpha_{m}(1+h) + aL \begin{cases} (1+h)\cos\theta \\ -\cos(\alpha_{m}+\theta) \\ -h\cos(e^{2}\alpha - \theta) \end{cases} \\ + 2bL(X^{*} + h^{2}X_{1}^{*}) \\ + 2\pi t f_{y}L^{2}(Y^{*} + h^{2}Y_{1}^{*}) \end{bmatrix}$$
(26)

• Mean Crushing Load

Assuming that the energy dissipation in the axisymmetric axial crushing of frusta takes place in the form of flexural and circumferential deformation, therefore, the external work done can be equated to the sum of energy absorbed in bending and circumferential stretching. Total work done by applied load is given by

$$W_{p} = P_{m} \delta_{T}$$
⁽²⁷⁾

where, δ_{τ} is the effective crushing distance corresponding to one complete fold

$$\delta_T = \left[L(1+h) - t - \frac{bL}{\alpha_m} \left(1 + \frac{h}{e} \right) \right] \cos\theta \qquad (28)$$

Equating the work done by the applied load given by Eqn (27) to the total energy absorbed in the formation of fold given by Eqn (26), the mean crushing load, therefore, can be calculated as

$$\frac{t}{2\sqrt{3}} \begin{cases} 2R\alpha_{m}(1+h) \\ +aL \begin{cases} (1+h)\cos\theta - \cos(\alpha_{m}+\theta) \\ -h\cos(e^{2}\alpha_{m}-\theta) \\ +2bL(X^{*}+hX_{1}^{*}) \end{cases} \end{cases}$$

$$\frac{P_{m}}{P_{o}} = \frac{+2\pi t f_{y}L^{2}(Y^{*}+h^{2}Y_{1}^{*})}{R \left[L(1+h)-t-\frac{bL}{\alpha_{m}}\left(1+\frac{h}{e}\right)\right]\cos\theta}$$
(29)

where
$$P_o = 2\pi R t f_v$$
 (30)

• Size of Fold and Folding Parameter

Determination of size of fold and folding parameter requires the minimisation of mean crushing load wrt fold length and folding parameter, respectively. Thus, the final expression for fold length is given by

$$L = \frac{E_1 - E_2}{E_3}$$
(32)

where

$$E_1 = P_m \left[L(1+h) - \frac{2bh}{e\alpha_m} \right] \cos\theta$$
(33)

$$E_{2} = \frac{\pi f_{y} t^{2}}{\sqrt{3}} \begin{bmatrix} a \left\{ (1+h)\cos\theta - \cos(\alpha_{m} + \theta) - h\cos(e^{2}\alpha_{m} - \theta) \right\} \\ -h\cos(e^{2}\alpha_{m} - \theta) \\ -2ae^{2}h\alpha_{m}(1-eh) \\ \sin(e^{2}\alpha_{m} - \theta) + 2b(X^{*} + hX_{1}^{*}) \\ +4bh(1-eh)\left[\cos(e^{2}\alpha_{m} - \theta) - \cos(e^{2}\alpha_{m} - \theta)\right] \end{bmatrix}$$
(34)

$$E_{3} = 4\pi t f_{y} \left(Y^{*} + h^{2} Y_{1}^{*}\right) + 2\pi f_{y} t h^{2} \left(1 - eh^{2}\right)$$

$$\left[2m(1-m)\left\{\sin\left(e^{2}\alpha_{1m} - \theta\right) + \sin\theta\right\}$$

$$- 2b\left\{\sin\left(e^{2}\alpha_{1m} - \theta\right) + \sin\theta\right\}$$

$$+ \frac{4b}{e^{2}\alpha_{m}} \left\{b\cos\left(e^{2}\alpha_{m} - \theta\right)\right\}$$

$$+ 2\left\{b^{2}\sin\left(e^{2}\alpha_{m} - \theta\right) - m^{2}\sin\left(e^{2}\alpha_{1m} - \theta\right)\right\}$$

$$+ \frac{4b^{2}}{e^{4}\alpha_{m}} \left\{\sin\left(e^{2}\alpha_{1m} - \theta\right)\right\}$$

$$- \sin\left(e^{2}\alpha_{m} - \theta\right)\right\} - \left(a^{2} + 2b^{2}\right)e^{2}\alpha_{m}\cos\left(e^{2}\alpha_{m} - \theta\right)$$
(35)

The expression for folding parameter can thus be obtained from:

$$\frac{P_m bL(1-h)\cos\theta}{2\pi f_y t\alpha_m R} - \frac{tae^3(1-h)L\alpha_m \sin(e^2\alpha_m - \theta)}{\sqrt{3}R}$$
$$-\frac{bL}{\sqrt{3}} \left(\frac{\partial X^*}{\partial m} + h\frac{\partial X_1^*}{\partial m}\right) - L \left(\frac{\partial Y^*}{\partial m} + h^2\frac{\partial Y_1^*}{\partial m}\right) = 0$$
(36)

where

$$\frac{\partial X^*}{\partial m} = \frac{1}{m} \left\{ \cos(\alpha_{1m} + \theta) - \cos \theta \right\}$$
(37)

$$\frac{\partial X_{1}^{*}}{\partial m} = \frac{2eL(1-h)}{hR} \left\{ \cos\left(e^{2}\alpha_{1m} - \theta\right) - \cos\left(e^{2}\alpha_{m} - \theta\right) \right\} + \frac{1}{m} \left\{ \cos\left(e^{2}\alpha_{1m} - \theta\right) - \cos\theta \right\}$$
(38)

$$\frac{\partial Y^*}{\partial m} = (2m-1) \int_0^{\alpha_m} \frac{1}{\alpha} \sin(\alpha_1 + \theta) d\alpha + (2m-1) \\ \sin(\alpha_{1m} + \theta) - (m-1) \sin\theta$$
(39)

$$\frac{\partial Y_{1}^{*}}{\partial m} = (2m-1)\int_{0}^{\alpha} \frac{1}{\alpha} \sin(e^{2}\alpha_{1}-\theta)d\alpha + \begin{pmatrix} m+1\\ -\frac{2me}{h} \end{pmatrix}$$

$$\left\{ \sin(e^{2}\alpha_{m}-\theta) + \sin\theta \right\} + \frac{L(1-h)}{hR}$$

$$\left[2be\left\{ \frac{\sin\left(e^{2}\alpha_{m}\right)}{-\theta} \right\} - \frac{4b}{e\alpha_{m}} \left\{ \frac{b\cos\left(e^{2}\alpha_{m}\right)}{-\theta} \right\} - \frac{m\cos\left(e^{2}\alpha_{m}\right)}{-\theta} \right\}$$

$$+ 2e\left\{ \frac{m^{2}\sin\left(e^{2}\alpha_{1m}\right)}{-\theta} \right\} - \frac{4b^{2}}{e^{3}\alpha_{m}^{2}} \left\{ \sin\left(e^{2}\alpha_{1m}\right) - \frac{b^{2}\sin\left(e^{2}\alpha_{1m}\right)}{-\theta} \right\} - \frac{4b^{2}}{e^{3}\alpha_{m}^{2}} \left\{ \sin\left(e^{2}\alpha_{1m}\right) - \frac{b^{2}\sin\left(e^{2}\alpha_{m}\right)}{-\theta} \right\} + \left(a^{2}+2b^{2}\right)e^{3}\alpha_{m}\cos\left(e^{2}\alpha_{m}\right) \right\}$$

$$+ m\sin\left(e^{2}\alpha_{1m}-\theta\right)$$

$$(40)$$

• Maximum Hinge Angle

The expression for deformation is given by

$$\delta = (1+h)L\cos\theta + \frac{2bL}{\alpha} \\ \left\{ \left(1 - \frac{h}{e}\right)\sin\theta - \sin(\alpha + \theta) - \frac{h}{e}\sin(e\alpha - \theta) \right\} \\ - aL\{\cos(\alpha + \theta) + h\cos(e\alpha - \theta)\}$$
(41)

Now, when $\alpha = \alpha_m$ then $\delta = \delta_r$, therefore Eqn (41) reduces to the following equation from where α_m can be determined as

$$t\alpha_{m} \cos \theta + bL\left(1 + \frac{h}{e}\right) \cos \theta$$

+ $2bL\left\{\left(1 - \frac{h}{e}\right) \sin \theta - \sin\left(\frac{\alpha_{m}}{e}\right) - \frac{h}{e} \sin\left(\frac{e\alpha_{m}}{e}\right)\right\}$
- $aL\alpha_{m}\left\{\cos(\alpha_{m} + \theta) + h\cos(e\alpha_{m} - \theta)\right\} = 0$ (42)

2.2.2 Case 2 – Folding Parameter Lies between b and c

The model for axisymmetric crushing of the frustum for this case is shown in Fig. 2. The initial line of frusta cuts the straight portion of fold at A'.

• Incremental Energy Absorbed in Bending of Lower Limb

The mean radii at A and C are given by

$$R_{A} = R + \rho \{\cos(\alpha + \theta) - \cos\theta\} + (m - b)L\sin(\alpha + \theta)$$
(43)

$$R_{c} = R + \rho \{\cos\theta - \cos(\alpha + \theta)\} + (c - m)L \sin(\alpha + \theta)$$
(44)

Substituting the values of R_{λ} and R_{c} in Eqn (12), one obtains the energy absorbed in bending of lower limb for a small increment of angle $d\alpha$ as

$$dW_{b} = 2\pi M_{p} \left[2R + L(1-2m)\sin\binom{\alpha}{+\theta} \right] d\alpha$$
 (45)



Figure 2. Model for crushing of frusta for case 2 (case 2: m lies between b and c).

Incremental Energy Absorbed in Hoop Action of Lower Limb

The expressions for incremental energy due to hoop strain for various regions are as follow.

Region AB

$$dW_{1} = 2\pi t f_{y} L^{2} [b(m-b)\cos(\alpha + \theta) + \frac{b^{2} \sin(\alpha + \theta)}{\alpha} + \frac{b^{2} \cos(\alpha + \theta)}{\alpha^{2}} + \frac{b^{2}}{\alpha^{3}} \{\sin(\alpha_{1} + \theta) - \sin\theta\}] d\alpha$$
(46)

Region BA'

$$dW_2 = 2\pi t f_y L^2 \bigg[0.5 (m-b)^2 \cos(\alpha + \theta) \bigg] d\alpha \quad (47)$$

$$dW_{3} = 2\pi t f_{y} L^{2} \left[\dot{0.5} \left(c - m \right)^{2} \cos\left(\alpha + \theta \right) \right] d\alpha \quad (48)$$

Region B'C

$$dW_{4} = 2\pi t f_{y} L^{2} \begin{bmatrix} b(c-m)\cos(\alpha+\theta) \\ -\frac{b^{2}\cos(\alpha+\theta)}{\alpha^{2}} \\ +\frac{b^{2}}{\alpha^{3}} \left\{ \sin(\alpha_{1}+\theta) - \sin\theta \right\} \end{bmatrix} d\alpha$$
(49)

Total Incremental Energy Absorbed in Lower Limb

Total incremental energy for lower limb is obtained by adding Eqns (46) to (49), thus

$$dW_{T1} = \frac{\pi f_y t^2}{\sqrt{3}} [2R + L(1 - 2m)\sin(\alpha + \theta)] d\alpha$$
$$+ 2\pi f_y t L^2 \left[\frac{b^2}{\alpha} \sin(\alpha + \theta) + J\cos(\alpha + \theta) \right] d\alpha$$
(50)

where

$$J = ab + 0.5 \left\{ (m-b)^2 + (c-m)^2 \right\}$$
(51)

Total energy absorbed in full crushing of the lower limb, $W_{T1} = \int_{\alpha}^{\alpha} dW_{T1}$

$$= \frac{\pi f_{y}t^{2}}{\sqrt{3}} \left[2R\alpha_{m} + L(1-2m) \left\{ \cos\theta - \cos\left(\frac{\alpha_{m}}{+\theta}\right) \right\} \right] + 2\pi t f_{y}L^{2}Y^{*}$$
(52)

where

$$Y^* = b^2 \int_{0}^{\alpha_m} \frac{1}{\alpha} \sin(\alpha + \theta) d\alpha + J \begin{cases} \sin(\alpha_m - \theta) \\ -\sin\theta \end{cases}$$
(53)

• Total Energy Absorbed in Upper Limb

Total potential energy for upper limb can be obtained by multiplying $\alpha_1, \alpha_1, \alpha_m, \alpha_{1m}$ by e^2 and replacing L by hL and θ by $-\theta$ in the energy equation of lower limb. The final expression becomes

$$W_{T2} = \frac{\pi f_y t^2}{\sqrt{3}} \begin{bmatrix} 2hR\alpha_m + hL(1-2m) \\ \{\cos\theta - \cos(e^2\alpha_m - \theta)\} \end{bmatrix} + 2\pi t f_y h^2 L^2 Y_1^*$$
(54)

where

$$Y_{1}^{\bullet} = b^{2} \int_{0}^{\alpha} \frac{1}{\alpha} \sin \left(\frac{e^{2} \alpha}{-\theta} \right) d\alpha + J \begin{cases} \sin \left(e^{2} \alpha_{m} - \theta \right) \\ + \sin \theta \end{cases}$$
(55)

• Total Energy Absorbed in Formation of One Complete Fold

The total energy absorbed in the formation of one complete fold is thus given by

$$W_{T} = W_{T1} + W_{T2}$$

$$= \frac{\pi f_{y} t^{2}}{\sqrt{3}} \begin{bmatrix} 2R\alpha_{m} (1+h) + L (1+h)(1-2m) \\ \cos\theta - (1-2m)L \begin{cases} \cos(\alpha_{m} + \theta) \\ + h \cos(e^{2}\alpha - \theta) \end{cases} \end{bmatrix} \\ + 2\pi t f_{y} L^{2} (Y^{*} + h^{2} Y_{1}^{*})$$
(56)

Mean Collapse Load

The final expression to find the mean collapse load corresponding to one complete fold is obtained by equating the total internal energy to the total work done by the applied load as

$$\frac{t}{2\sqrt{3}} \begin{cases} 2R\alpha_{m}(1+h) + L(1-2m) \\ (1+h)\cos\theta - L(1-2m) \\ \left\{\cos(\alpha_{m}+\theta) + h\cos(e^{2}\alpha_{m}-\theta)\right\} \end{cases}$$

$$\frac{P_{m}}{P_{o}} = \frac{+L^{2}(Y^{*}+h^{2}Y_{1}^{*})}{R\left[L(1+h)-t-\frac{bL}{\alpha_{m}}\left(1+\frac{h}{e}\right)\right]\cos\theta}$$
(57)

• Size of Fold and Folding Parameter

The size of fold obtained by minimising P_m wrt L is given by

$$L = \frac{E_1 - E_2}{E_3}$$
(58)

where

$$E_{1} = \frac{P_{m}}{\pi f_{v} t} \left[(1+h) - \frac{2bh}{e\alpha_{m}} \right] \cos\theta$$
(59)

$$E_{2} = \frac{t(1-2m)}{\sqrt{3}} \begin{bmatrix} (1+h)\cos\theta - \cos(\alpha_{m}+\theta) \\ -h\cos(e^{2}\alpha_{m}-\theta) \\ -2e^{2}h\alpha_{m}(1-eh)\sin(e^{2}\alpha_{m}-\theta) \end{bmatrix}$$
(60)

$$E_{3} = 4\left(Y^{*} + h^{2}Y_{1}^{*}\right) - 4h^{2}(1 - eh)$$

$$\left[b^{2}\left\{\sin\left(e^{2}\alpha_{1m}\right) + \sin\theta\right\} + J\alpha_{m}e^{2}\cos\left(e^{2}\alpha_{m}\right)\right]$$
(61)

The expression for folding parameter can thus be obtained from

$$\frac{P_m bL(1-h)\cos\theta}{2\pi f_y t\alpha_m R} + \frac{t}{\sqrt{3}} \begin{cases} (1+h)\cos\theta - \cos\begin{pmatrix}\alpha_m\\+\theta\end{pmatrix} \\ -h\cos\left(e^2\alpha_m - \theta\right) \end{cases} \\ - L\left(\frac{\partial Y^*}{\partial m} + h^2\frac{\partial Y_1^*}{\partial m}\right) \\ + \frac{te^3(2m-1)(1-h)L\alpha_m\sin\left(e^2\alpha_m - \theta\right)}{\sqrt{3}R} = 0 \end{cases}$$

where

$$\frac{\partial Y^{*}}{\partial m} = \sin \theta - \sin (\alpha_{m} + \theta)$$
 (62)

$$\frac{\partial Y_1^*}{\partial m} = \frac{2eL(1-h)}{hR} \begin{bmatrix} b^2 \left[\sin\theta + \sin\left(e^2\alpha_m - \theta\right) \right] \\ + Je^2\alpha_m \cos\left(e^2\alpha_m - \theta\right) \end{bmatrix} \\ - \left\{ \sin\left(e^2\alpha_m - \theta\right) + \sin\theta \right\}$$
(63)

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Contributors



Mr MMA Khan obtained his MSc (Building Engg) from Aligarh Muslim University Aligarh, in 1999. Presently, he is Lecturer at the University Polytechnic, Aligarh Muslim University, Aligarh. His areas of interest are: Impact mechanics and structrual design.



Prof H Abbas obtained his MSc (Building Engg) from the Aligarh Muslim University, Aligarh, in 1984 and PhD from the Indian Institute of Technology Roorkee, in 1993. Presently, he is working as Professor of Civil Engineering at the Aligarh Muslim University. He has published/presented 50 research papers in national/international journals and conferences. He is the recipient of *Khosla Award* and *Khosla Research Prize* of IIT Roorkee for his two research papers He has developed many software packages for civil engineering applications. His areas of research includes: Impact mechanics, structural reliability and structrual dynamics. He is working on many sponsored research projects.